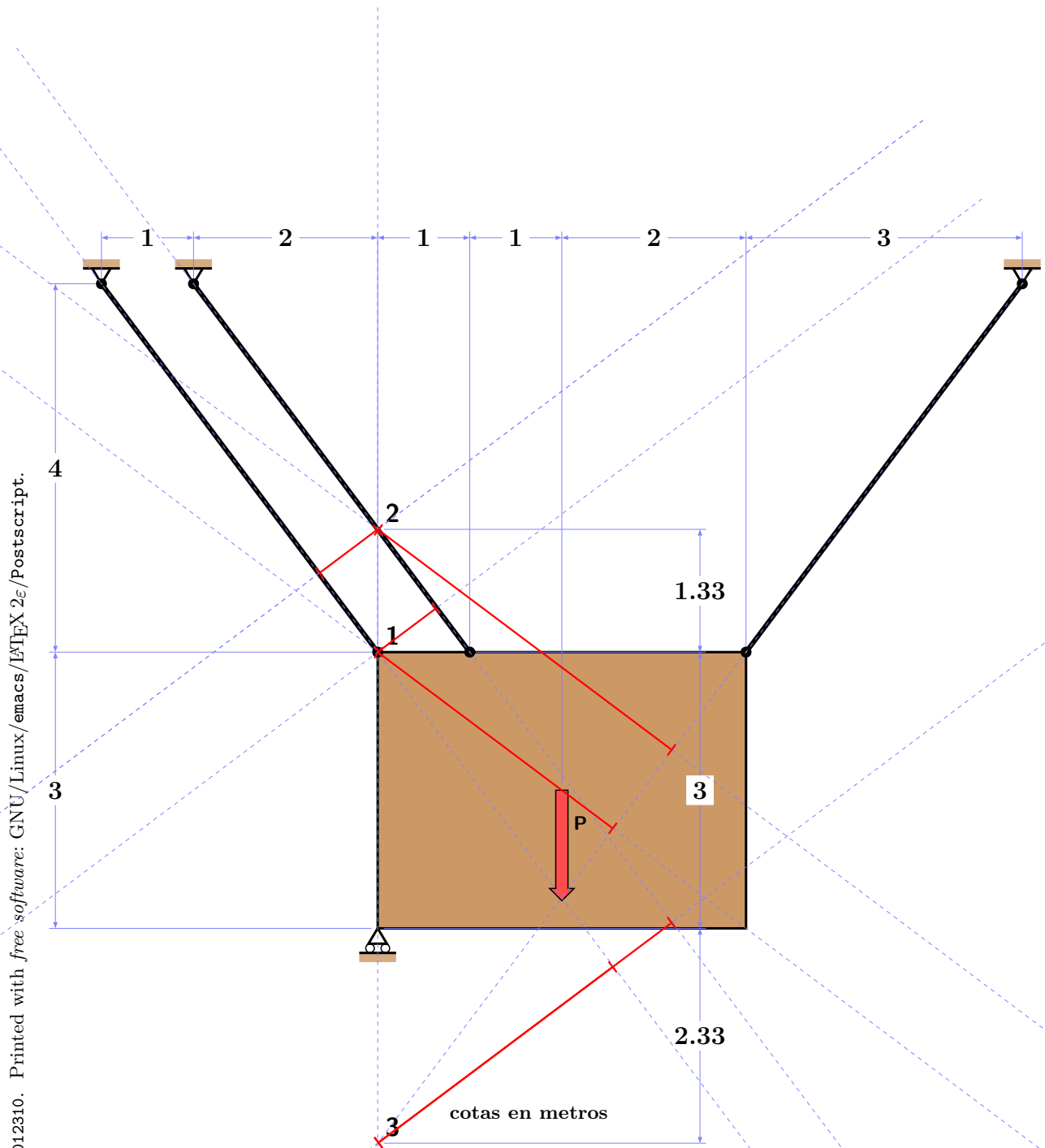
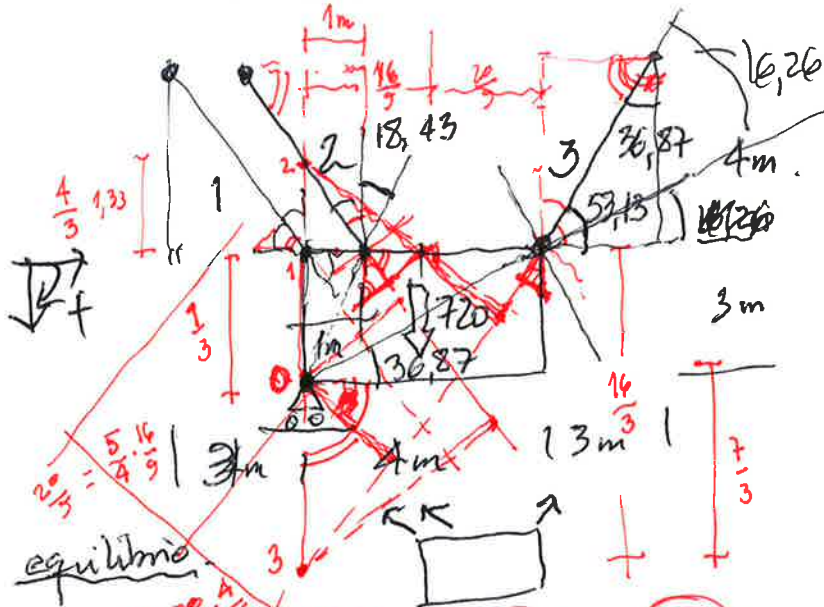


Estructura con dos grados de libertad y tres modos de deformación





~~1/2~~

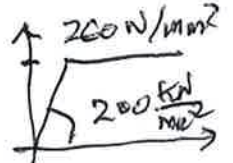
barras 2φ40 mm

$$A = 2 \times 1257 \text{ mm}^2 = 2513 \text{ mm}^2$$

$$K = \frac{2513 \text{ mm}^2 \times 200 \text{ kN/mm}^2}{5000 \text{ mm}}$$

$$= 101 \frac{\text{kN}}{\text{mm}}$$

$$N_u = 653 \text{ kN}$$



$$\begin{cases} \epsilon_u = 10 \text{ mm/m} \\ \delta_u = 50 \text{ mm} \end{cases}$$

equilibrio

$$0 = -N_1 \frac{3}{5} - N_2 \frac{3}{5} + N_3 \frac{3}{5} = 0$$

$$1440 \text{ kN} - N_1 \cdot \frac{3}{5} \times 3\text{m} - N_2 \left(\frac{3}{5} \times 3\text{m} + \frac{4}{5} \times 4\text{m} \right) + N_3 \left(\frac{3}{5} \times 3\text{m} - \frac{4}{5} \times 4\text{m} \right) = 0$$

$$\begin{bmatrix} 0 \\ 1440 \end{bmatrix} = \underbrace{\begin{bmatrix} 3/5 & 3/5 & -3/5 \\ 9\text{m}/5 & 13\text{m}/5 & 7\text{m}/5 \end{bmatrix}}_{B^T} \begin{bmatrix} N_1 \\ N_2 \\ N_3 \end{bmatrix}$$

compatibilidad

	u	θ
δ_1	$3/5$	$3\text{m} \times 3/5$
δ_2	$3/5$	$3,16\text{m} \times 0,82$
δ_3	$-3/5$	$5\text{m} \times 0,28$

$$\begin{bmatrix} \delta_1 \\ \delta_2 \\ \delta_3 \end{bmatrix} = \underbrace{\begin{bmatrix} 3/5 & 9\text{m}/5 \\ 3/5 & 13\text{m}/5 \\ -3/5 & 7\text{m}/5 \end{bmatrix}}_B \begin{bmatrix} u \\ \theta \end{bmatrix}$$

rigidez de elementos

$$K_i = 101 \frac{\text{kN}}{\text{mm}}$$

$$\begin{bmatrix} N_1 \\ N_2 \\ N_3 \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \times 101 \frac{\text{kN}}{\text{mm}} \times \begin{bmatrix} \delta_1 \\ \delta_2 \\ \delta_3 \end{bmatrix}$$

SOLUCIÓN PERÍODO PROPORCIONAL.

$$a = \begin{bmatrix} -2,65 \text{ mm} \\ 1,59 \text{ mm/m} \end{bmatrix} \quad S = \begin{bmatrix} 129 \text{ kN} \\ 257 \text{ kN} \\ 386 \text{ kN} \end{bmatrix} \quad \sigma = \begin{bmatrix} 51,2 \\ 102 \\ 153 \end{bmatrix} \frac{\text{N}}{\text{mm}^2}$$

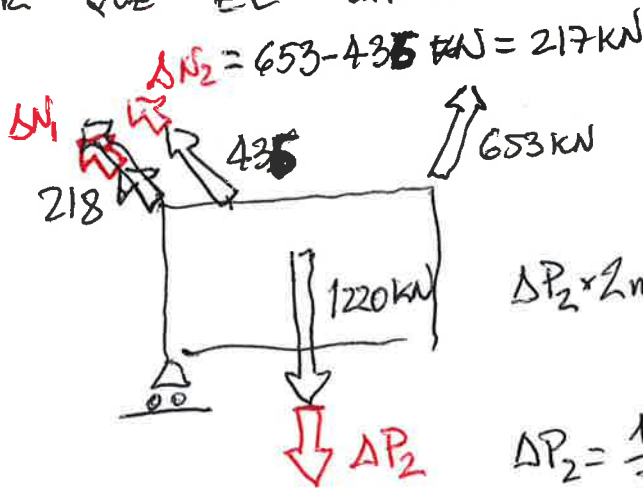
LÍMITE ELÁSTICO.

$$\min_i \left(\frac{\sigma_e}{\sigma_i} \right) = \frac{260}{153} = 1,69. \quad \boxed{P_e = 1.220 \text{ kN}} = P_1$$

$$= \frac{653}{386} = \frac{653}{386}$$

$$a = \begin{bmatrix} -4,49 \text{ mm} \\ 2,69 \text{ mm/m} \end{bmatrix} \quad S = \begin{bmatrix} 218 \\ 436 \\ 653 \end{bmatrix} \text{ kN}$$

CARGA MAYOR QUE EL LÍMITE ELÁSTICO.



análisis isostático.

$$\Delta N_1 = -\Delta N_2 \quad (\sum F_H = 0)$$

$$\Delta P_2 \times 2 \text{ m} = \Delta N_1 \times \frac{9 \text{ m}}{5} + \Delta N_2 \times \frac{13 \text{ m}}{5}$$

($\sum M = 0$)

$$\Delta P_2 = \frac{1}{2} \Delta N_2 \left(\frac{4}{5} \right) = \frac{2}{5} 217 \text{ kN} = 86,8 \text{ kN}$$

$$\delta_1 = 0,01 \text{ mm}$$

$$\delta_2 = 6,47 \text{ mm}$$

$$\begin{bmatrix} \delta_1 \\ \delta_2 \end{bmatrix} = \begin{bmatrix} 0,60 & 1,8 \text{ m} \\ 0,60 & 2,6 \text{ m} \end{bmatrix} \begin{bmatrix} a \\ \sigma \end{bmatrix}$$

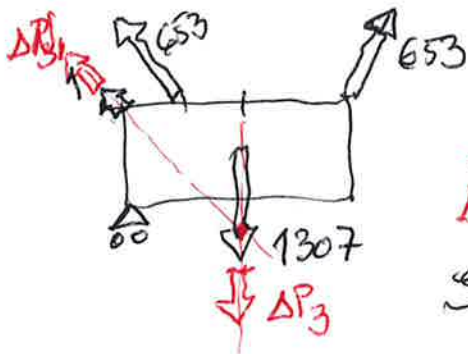
$$\begin{bmatrix} a \\ \sigma \end{bmatrix} = \begin{bmatrix} -24,21 \text{ mm} \\ 8,08 \text{ mm/m} \end{bmatrix}$$

$$\delta_3 = 25,83 \text{ mm} < \delta_M = 50 \text{ mm}$$

$$\Delta N_1 = -217 \text{ kN}$$

$$\begin{bmatrix} N_1 = 1 \text{ kN} \\ N_2 = 653 \text{ kN} \\ N_3 = 653 \text{ kN} \end{bmatrix} \quad P_2 = 1307 \text{ kN}$$

¿CARGA MAYOR QUE P_2 ?



$\vec{\Delta P}_3 + \vec{\Delta N}_1 \neq 0$
equilibrio imposible.

ROTURA FÍSICA.

$\delta_1 = 0,01 \text{ mm.}$

$\delta_3 = 50 \text{ mm.}$

$$\begin{cases} \delta_1 \\ \delta_3 \end{cases} = \begin{bmatrix} 0,60 & 1,80 \text{ m} \\ -0,60 & 1,40 \text{ m} \end{bmatrix} \begin{bmatrix} u \\ \theta \end{bmatrix}$$

$$\begin{cases} u \\ \theta \end{cases} = \begin{bmatrix} -46,9 \text{ mm} \\ 15,6 \text{ mm/m.} \end{bmatrix}$$

($\text{arc tan } 0,0466 \approx 0,016$)
 $\text{tan } 0,016 \approx 0,016$)

$\delta_2 = 12,5 \text{ mm.}$

ANÁLISIS DIRECTO DE LA ROTURA. $g=2$ $s=3$ $g-1=1$

$\delta_3 = 0$

$-\frac{3}{5}u + 5 \text{ m} \times 0,22 \times 1,4 \text{ m} \theta = 0 \quad u = 2,33 \theta.$

$\begin{bmatrix} du \\ d\theta \end{bmatrix} = \begin{bmatrix} 2,33 \text{ m} \\ 1 \end{bmatrix} d\theta.$

$$\begin{bmatrix} d\delta_1 \\ d\delta_2 \\ d\delta_3 \end{bmatrix} = \begin{bmatrix} 3,2 \text{ m} \\ 4 \text{ m} \\ 0 \end{bmatrix} d\theta.$$

$W_{\text{ext}} = P_u \times 2 \text{ m} \times d\theta$

$W_{\text{int}} = 653 \text{ kN} (3,2 \text{ m} + 4 \text{ m}) d\theta$

$W_{\text{ext}} = W_{\text{int}}$

$P_u = \frac{1}{2 \text{ m}} 653 \text{ kN} (3,2 \text{ m} + 4 \text{ m}) = 2351.$

$d\delta_2 = 0$

$0,6 du + 2,6 d\theta = 0 \quad du = -4,33 m d\theta$

$\begin{bmatrix} du \\ d\theta \end{bmatrix} = \begin{bmatrix} -4,33 m \\ 1 \end{bmatrix} d\theta$

$dW_{ext} = dW_{int}$

$\begin{bmatrix} d\delta_1 \\ d\delta_2 \\ d\delta_3 \end{bmatrix} = \begin{bmatrix} -0,8 \\ 0 \\ 4 \end{bmatrix} d\theta$

~~$P_u \times 2m d\theta = 653 kN (-0,8m + 4m) d\theta$~~
 ~~$P_u = 1044 kN$~~

$P_u \times 2m d\theta = \{ 0 kN (-0,8m) + 653 kN \times 4m \} d\theta$

$P_u = 1306 kN$

$d\delta_4 = 0$

~~$-0,6 du + 1,4 d\theta$~~
 $0,6 du + 1,8 d\theta = 0 \quad du = -3 d\theta$

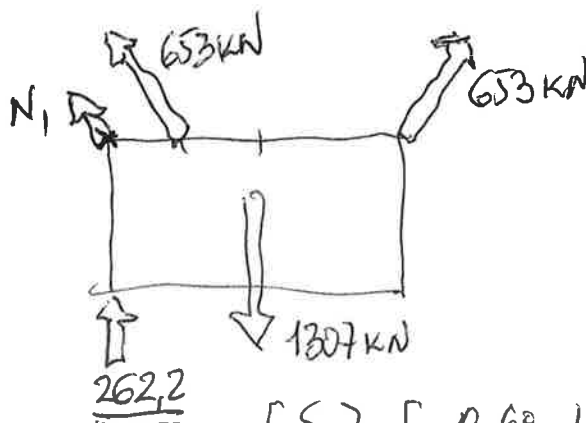
$\begin{bmatrix} du \\ d\theta \end{bmatrix} = \begin{bmatrix} -3 m \\ 1 \end{bmatrix} d\theta$

$\begin{bmatrix} d\delta_1 \\ d\delta_2 \\ d\delta_3 \end{bmatrix} = \begin{bmatrix} 0 \\ 0,8 m \\ 3,2 m \end{bmatrix} d\theta$

$P_u \times 2m d\theta = 653 kN \{ 0,8m + 3,2m \} d\theta$

$P_u = 2 \times 653 kN = 1306 kN$

ANÁLISIS DE LA SITUACIÓN DE EQUILIBRIO. $d\delta_1 = 0$



$\sum F_H = 0 \Rightarrow N_1 = 0$

$\sum F_V = 0 \Rightarrow R = 262,2$

$\delta_1 = 0$

$(\delta_2 \text{ o } \delta_3) = 6,47 mm$

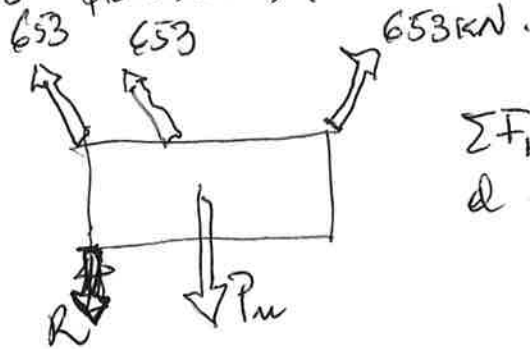
2 la vista de $[B] \times [C]$

o $\delta_2 = 6,47 mm$

$\begin{bmatrix} \delta_1 \\ \delta_2 \end{bmatrix} = \begin{bmatrix} 0,60 & 1,8 m \\ 0,60 & 2,6 m \end{bmatrix} \begin{bmatrix} u \\ \theta \end{bmatrix} \Rightarrow \begin{bmatrix} u \\ \theta \end{bmatrix} = \begin{bmatrix} -24 mm \\ 8,09 mm/m \end{bmatrix}$

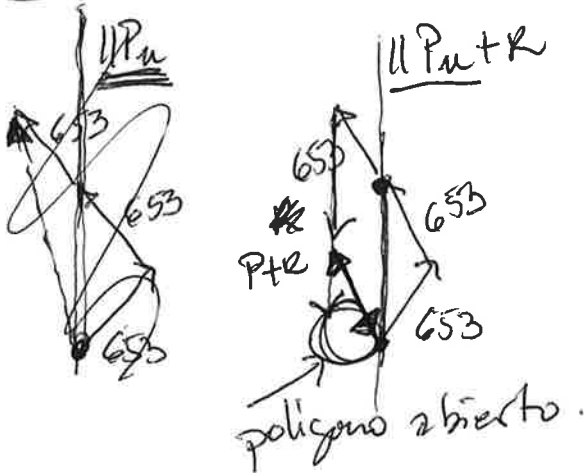
$\delta_3 = 25,88 mm/m$

MECANISMO DE COLAPSO CON $S_K=0$
(todos los cables plastifican).

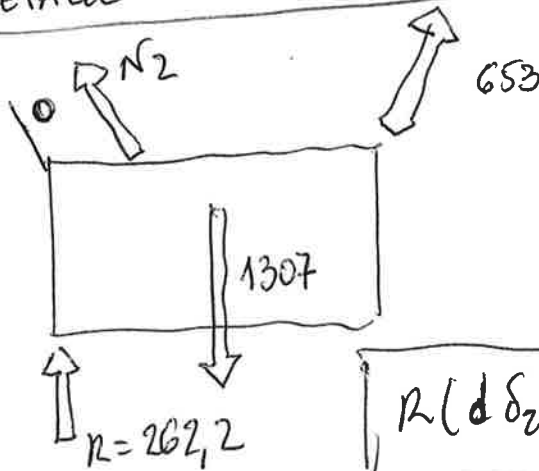


$\Sigma F_H \neq 0$
de equilibrio sería imposible.

LA CUENTA DE LA VIEJA.



UN ANÁLISIS DE DETALLE PARA $d\delta_2=0$

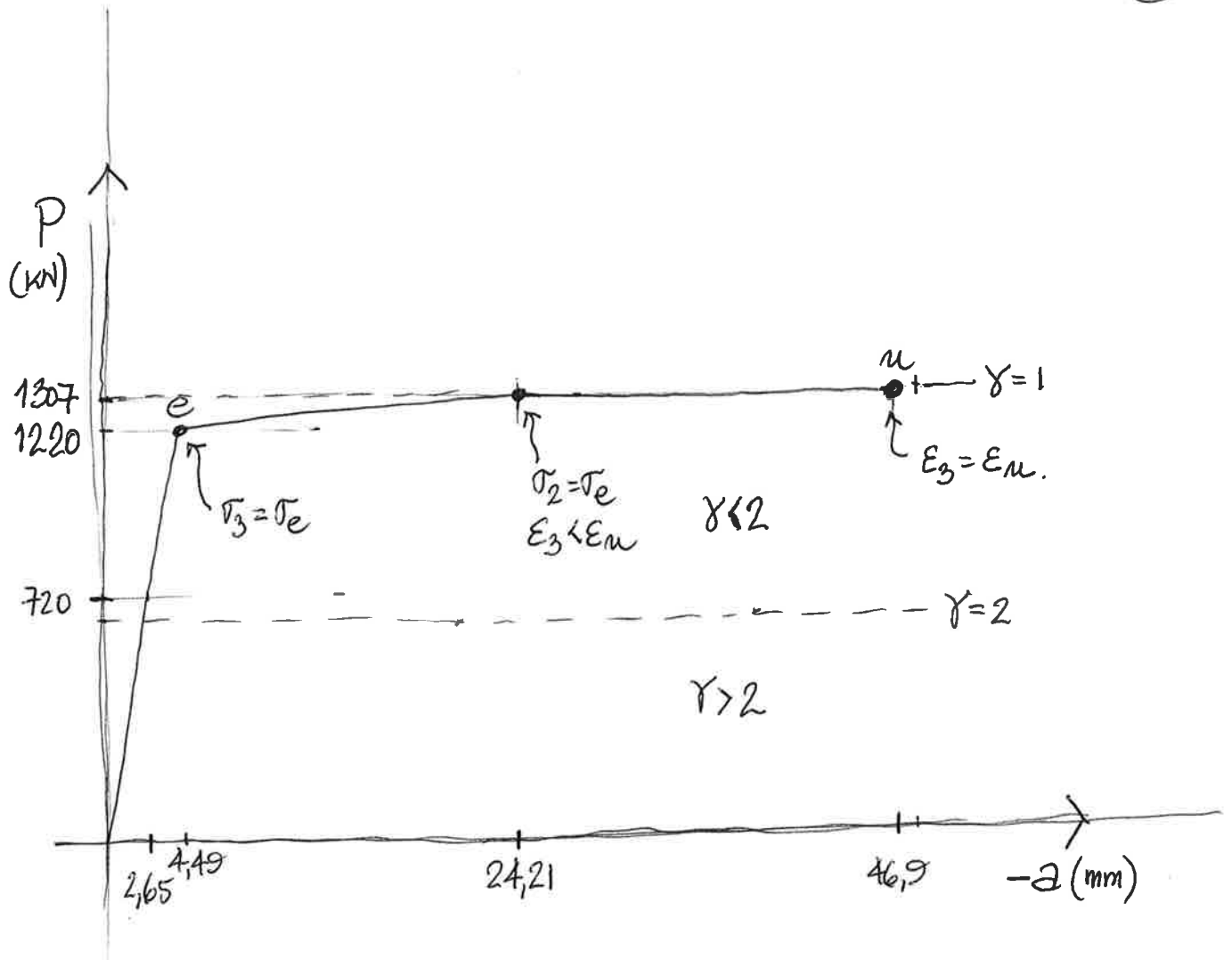


$\Sigma F_H = 0 \Rightarrow N_2 = 653$
 $\Sigma F_V = 0 \Rightarrow R = 262,2$

$R(d\delta_2=0) \equiv R(d\delta_1=0)$

SOLIDO DEFORMABLE (II). EJEMPLO COMPLICADO

6

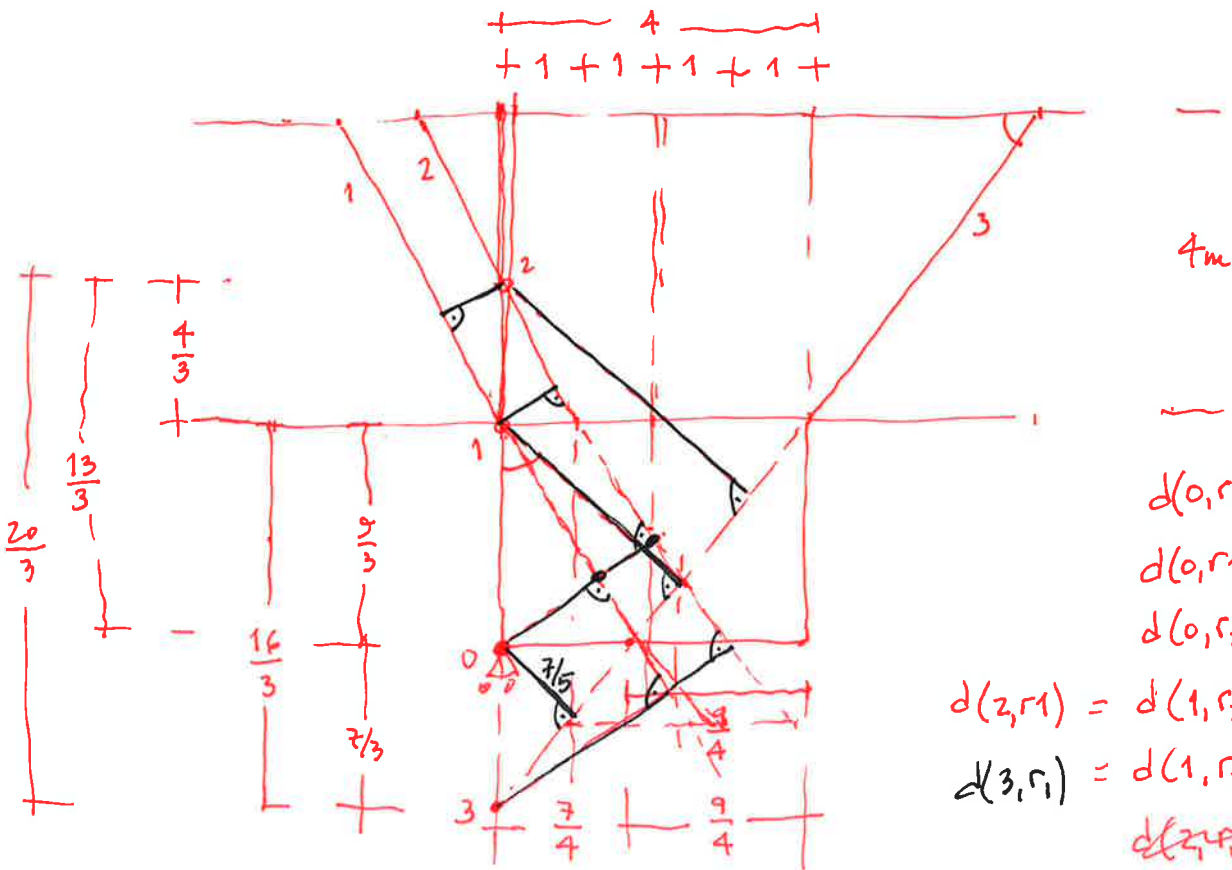
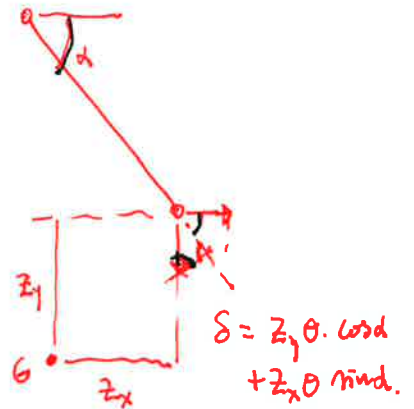
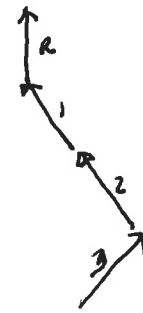


$$\begin{cases} \delta_1 \\ \delta_2 \\ \delta_3 \end{cases} = \begin{bmatrix} 0 & -\frac{4}{3}m \times \frac{3}{5} & \frac{16}{3}m \times \frac{3}{5} \\ 1m \cdot \frac{4}{5} & 0 & \frac{20}{3}m \times \frac{3}{5} \\ 4m \cdot \frac{4}{5} & 4m & 0 \end{bmatrix} \begin{cases} \theta_1 \\ \theta_2 \\ \theta_3 \end{cases}$$

$g=2$
 $s=3$

$$\begin{bmatrix} 0 & -0,8m & 3,20m \\ 0,8m & 0 & 4m \\ 3,20m & 4m & 0 \end{bmatrix}$$

$$\begin{cases} \delta_1 \\ \delta_2 \\ \delta_3 \end{cases} = \begin{bmatrix} 0 & -0,8m & 3,20m & 1,8m \\ 0,8m & 0 & 4m & 2,60m \\ 3,20m & 4m & 0 & 1,40m \end{bmatrix} \begin{cases} \theta_1 \\ \theta_2 \\ \theta_3 \\ \theta_4 \end{cases}$$



$d(0, r_3) = \frac{7}{5} = 1,20$

$d(0, r_1) = \frac{9}{5} = 1,8$

$d(0, r_2) = \frac{13}{5} = 2,60$

$d(2, r_1) = d(1, r_2) = \frac{4}{5} = 0,8m$

$d(3, r_1) = d(1, r_3) = \frac{16}{5} = 3,20m$

$d(2, r_1) = \frac{4}{5} = 0,8$

$d(3, r_2) = d(2, r_3) = \frac{20}{5} = 4m$

$d(3, r_1)$